

Net Positive Suction Head - - NPSHr & NPSHa

[Joe Evans, Ph.D](#)

<http://www.pumped101.com>

In the January 2007 edition of P&S, I wrote an article entitled "When Bubbles Don't Burst". It was about cavitation and showed how a collapsing water vapor bubble can damage an impeller. Since then, I have received a number of requests to address Net Positive Suction Head (NPSH) and its relationship to cavitation. Here it is in a very simple, Pump Ed 101 perspective.

The process of boiling is not as simple as it may seem. We tend to think that it is all about temperature and often forget that pressure has an equal role in the process. The point at which water boils is proportional to both its temperature and the pressure acting upon its surface. As pressure decreases, so does the temperature required to initiate boiling. The onset of cavitation also follows this rule. When water, at some ambient temperature, travels through an area of low pressure it can undergo a change in state from liquid to vapor (boiling). As it progresses into an area of higher pressure it will return to the liquid state (cavitation). The bubbles that form and collapse during this process are those of water vapor - - not air. Although dissolved or entrained air can affect pump performance, it produces a totally different kind of bubble than the one produced by boiling.

The fact that boiling is proportional to both temperature and pressure is the reason cavitation is such a persistent problem. Simply stated, water can boil at virtually any temperature. At sea level, where atmospheric pressure is about 14.7 PSI (34'), it takes 212 deg F. Increase that elevation to 6000' and it drops to around 200 deg F because the corresponding atmospheric pressure decreases to 11.7 PSI (27'). If we introduce a vacuum and continue to reduce pressure to about 0.2', it will boil at its freezing point. Well, so what? We don't usually operate a pump in a vacuum and even at the top of Mt Everest we still have almost 5.2 PSI (12') of atmospheric pressure!

Well, it turns out that all centrifugal pumps produce a partial vacuum. If they did not they would be unable to pump water from a lower level. During normal operation, the area of lowest pressure occurs near the impeller vane entrances and

if the pressure in this area drops to about 1', water will boil at 75 deg F! In order for a pump to operate cavitation free, an excess of pressure energy is required of the water entering this area. We typically refer to this requirement as NPSHr - - or the NPSH required. Where does this pressure energy come from? It is a combination of several different forms of energy that exist, at various levels, on the suction side of the pumping system. We refer to this available pressure energy as NPSHa - - or the NPSH available.

NPSHa

The **NPSH available** to a centrifugal pump combines the effect of atmospheric pressure, water temperature, supply elevation, and the dynamics of the suction piping. The following equation illustrates this relationship. All values are in feet of water and the sum of these components represents the total pressure available at the pump suction.

$$\text{NPSHa} = H_a \pm H_z - H_f + H_v - H_{vp}$$

Where:

H_a is the atmospheric or absolute pressure

H_z is the vertical distance from the surface of the water to the pump centerline

H_f is the friction formed in the suction piping

H_v is the velocity head at the pump's suction

H_{vp} is the vapor pressure of the water at its ambient temperature

H_a is the atmospheric or absolute pressure exerted on the surface of the water supply. Atmospheric pressure is the pressure due to the density of the earth's atmosphere at some elevation. It develops its greatest pressure (14.7 PSI) at sea level (were it is most dense) and approaches zero at its upper boundary. We seldom think about this pressure because, out of the box or on the work bench, the typical pressure gauge reads 0 PSI. These gauges are calibrated to something we call "gauge" scale (PSIG) and totally ignore atmospheric pressure. Gauges calibrated to the "absolute" scale (PSIA)



include atmospheric pressure and will read 14.7 PSI at sea level. Figure 1 compares these two pressure scales. On the absolute scale, 0 PSI equates to a perfect vacuum but on the gauge scale it equates to atmospheric pressure.

If the water source is a reservoir or an open (or vented) tank, H_a is simply the measured atmospheric pressure. It takes on another dimension if the supply is an enclosed, unvented tank. In this case H_a becomes the absolute pressure or the sum of the measured atmospheric pressure plus or minus the actual gauge pressure of the air in the tank.

H_z takes into account the positive or negative pressure of the water source due to its elevation. If it is above the pump H_z is a positive number and if it is below, H_z is negative. H_f is simply the friction generated due to flow in the suction piping and is always a negative number. It is a function of the pipe length and diameter plus the fittings and valves it incorporates.

H_v and H_{vp} may be a little less familiar to some of us. H_v , or velocity head, is the kinetic energy of a mass of water moving at some velocity V . It is the equivalent of the distance that water would have to fall in order to reach that velocity. It can be calculated by determining the velocity in the suction piping from a velocity table and substituting that value for V in the equation " $h = V^2/2g$ " (where g is the universal gravitational constant @ 32 ft/sec²). It is usually small - - at a velocity of 7 fps, H_v is just 0.765' - - and is often ignored if H_a and H_z are sufficiently large. H_{vp} represents the pressure that is required to keep water in the liquid state at some ambient temperature and is obtained from a vapor pressure table. At 50 deg F, just 0.41' is required but at 160 deg F that requirement increases to 11.2'. Since this pressure must be reserved for its stated purpose, H_{vp} is always a negative number.

At first glance, the equation for NPSHa looks pretty static but, it is actually quite dynamic. All of the variables can be in a continuous state of change. Velocity head and suction line friction vary as a function of flow. Likewise, atmospheric pressure can vary by several feet depending upon weather conditions. And, water supply elevation and temperature can vary seasonally. Usually the "worst case" values for each of these components are used when calculating NPSHa.

NPSHr

As I said earlier NPSHr is the suction pressure necessary to insure proper pump operation. It is purely a function of the pump design and, although it can be

calculated, it is more accurately determined by actual testing. Why does a pump require a positive suction head? Quite simply, it is impossible to design a centrifugal pump that exhibits absolutely no pressure drop between the suction inlet and its minimum pressure point, which normally occurs at the entrance to the impeller vanes. Therefore, all pump systems must maintain a positive suction pressure that is sufficient to overcome this pressure drop. If the pressure is not sufficient, some of the water will change state (liquid to vapor) and cavitation occurs. Like NPSHa, NPSHr is also a dynamic quantity and increases substantially with pump flow.

You would think that the NPSHr, measured by the pump manufacturer, would be the suction pressure required to prevent cavitation. That used to be the definition but, it is currently defined as the suction pressure at which a particular pump's hydraulic performance is degraded by 3%. This raises some concern since this degradation is actually due to cavitation and, at the 3% level, it has the potential to be damaging. The Hydraulic Institute's standards stipulate that each of the points on a pump manufacturer's NPSHr curve must reflect this 3% value. There are rumors that the term NPSHr will eventually be changed to NPSH₃ which more accurately describes its true meaning. Depending upon the pump design, HI recommends an NPSHa / NPSHr margin of 1.1 to 2.5. Some pump experts recommend even more. It is a good idea to check with your pump manufacturer for their specific margin requirement as it relates to a particular pump model and its application.

In recent years a new term, [NPSHi](#) (inception), was developed to define the suction pressure required that will suppress all cavitation. The cavitation that occurs between NPSHi and the point where damage occurs is called incipient cavitation. This form of cavitation appears to cause little, if any, damage in normal pumping applications. There is some ongoing debate as to whether the cavitation that occurs due to a 3% performance degradation should be regarded as incipient cavitation.

Joe Evans is the western regional manager for Hydromatic Engineered Waste Water Systems, a division of Pentair Water, 740 East 9th Street Ashland, OH 44805. He can be reached at joe.evans@pentairwater.com, or via his website at www.pumped101.com. If there are topics that you would like to see discussed in future columns, drop him an email.